

# A Nonlinear Fuel Optimal Reaction Jet Control Law

*E.F. Breittfeller, L.C. Ng*

This article was submitted to  
11<sup>th</sup> Annual American Institute of Aeronautics and  
Astronautics/Missile Defense Agency Technology Conference and  
Exhibit, Monterey, CA, July 29 – August 2, 2002

**June 30, 2002**

***U.S. Department of Energy***

Lawrence  
Livermore  
National  
Laboratory

## DISCLAIMER

This document was prepared as an account of work sponsored by an agency of the United States Government. Neither the United States Government nor the University of California nor any of their employees, makes any warranty, express or implied, or assumes any legal liability or responsibility for the accuracy, completeness, or usefulness of any information, apparatus, product, or process disclosed, or represents that its use would not infringe privately owned rights. Reference herein to any specific commercial product, process, or service by trade name, trademark, manufacturer, or otherwise, does not necessarily constitute or imply its endorsement, recommendation, or favoring by the United States Government or the University of California. The views and opinions of authors expressed herein do not necessarily state or reflect those of the United States Government or the University of California, and shall not be used for advertising or product endorsement purposes.

This is a preprint of a paper intended for publication in a journal or proceedings. Since changes may be made before publication, this preprint is made available with the understanding that it will not be cited or reproduced without the permission of the author.

This report has been reproduced directly from the best available copy.

Available electronically at <http://www.doc.gov/bridge>

Available for a processing fee to U.S. Department of Energy  
And its contractors in paper from  
U.S. Department of Energy  
Office of Scientific and Technical Information  
P.O. Box 62  
Oak Ridge, TN 37831-0062  
Telephone: (865) 576-8401  
Facsimile: (865) 576-5728  
E-mail: [reports@adonis.osti.gov](mailto:reports@adonis.osti.gov)

Available for the sale to the public from  
U.S. Department of Commerce  
National Technical Information Service  
5285 Port Royal Road  
Springfield, VA 22161  
Telephone: (800) 553-6847  
Facsimile: (703) 605-6900  
E-mail: [orders@ntis.fedworld.gov](mailto:orders@ntis.fedworld.gov)  
Online ordering: <http://www.ntis.gov/ordering.htm>

OR

Lawrence Livermore National Laboratory  
Technical Information Department's Digital Library  
<http://www.llnl.gov/tid/Library.html>

# A Nonlinear Fuel Optimal Reaction Jet Control Law

Eric F. Breitfeller\* and Lawrence C. Ng†

Lawrence Livermore National Laboratory  
P.O. Box 808, L-278, Livermore, CA 94551

## Abstract

We derive a nonlinear fuel optimal attitude control system (ACS) that drives the final state to the desired state according to a cost function that weights the final state angular error relative to the angular rate error. Control is achieved by allowing the pulse-width-modulated (PWM) commands to begin and end anywhere within a control cycle, achieving a pulse width pulse time (PWPT) control. We show through a MATLAB® Simulink model that this steady-state condition may be accomplished, in the absence of sensor noise or model uncertainties, with the theoretical minimum number of actuator cycles. The ability to analytically achieve near-zero drift rates is particularly important in applications such as station-keeping and sensor imaging. Consideration is also given to the fact that, for relatively small sensor and model errors, the controller requires significantly fewer actuator cycles to reach the final state error than a traditional proportional- integral-derivative (PID) controller. The optimal PWPT attitude controller may be applicable for a high performance kinetic energy kill vehicle.

## Introduction

In problems related to the attitude control of rigid bodies, it is often advantageous to design an attitude control system that is optimal in some way relative to a set of system constraints. Most often cited are constraints of time and fuel. A constraint pertinent to some systems regards the total number of PWM commands sent to the actuators. In our case that is represented by valve commands sent to the ACS jets. A PID controller is simple to implement (Wie<sup>1</sup>, 1998), takes few CPU cycles, and usually is robust in most applications. Sliding mode controllers are likewise robust and easy to implement (Wertz<sup>2</sup>, 1978). Both of these control systems share a potential common trait, in an environment where sensor noise exists, of requesting numerous PWM commands while dithering about their phase-plane trajectories. These effects can be minimized by introducing deadbands and other nonlinear elements at the expense of system response and accuracy.

We derive a single-axis PWPT ACS that minimizes the total number of valve commands while simultaneously minimizes a cost function dependent on the final state errors. We first apply the single-axis controller to a 1-DOF simulated system, and then to a 3-DOF system, where each axis is controlled by the single-axis controller. This allows us to evaluate its effectiveness in the presence of coupling through the inertia matrix. Phase-plane plots of angle versus angle rate are given as well as plots of the valve commands. Monte Carlo runs are made using Simulink in order to investigate controller sensitivities to gyro noise and camera noise. Published results (Garcia<sup>3</sup>, 1998) are shown in work done previously in support of MicroSat pointing and control development at LLNL.

## Pulse Width Modulation (PWM)

An approximation to an idealized linear command may be represented as in Figure-1, and is shown to start at the beginning of the control cycle and to end within the control cycle, or it may continue into the next cycle if the command is saturated.

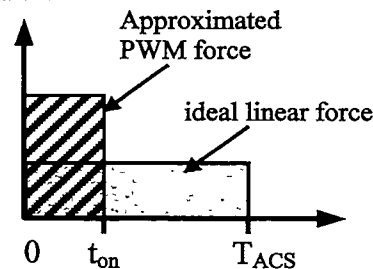


Figure-1: Ideal linear force command shown with PWM approximation

A discrete approximation to the ideal jet turn on time request can be made as follows,

$$t_{on} = \begin{cases} \frac{F_{cmd}}{F_{jet}} \cdot T_{ACS} & ; (F_{cmd} \leq F_{jet}) \\ \text{sign}(F_{cmd}) \cdot T_{ACS} & ; (F_{cmd} > F_{jet}) \end{cases}$$

Figure-2 illustrates a single-axis PWM control loop that provides feedback on error and error rate.

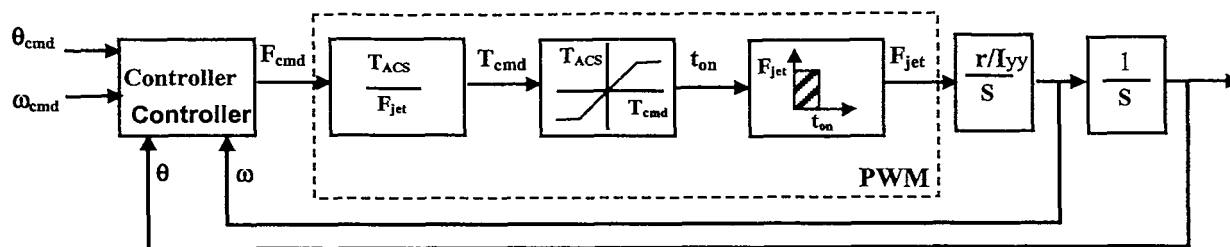


Figure-2: Simplified single-axis controller

## Nonlinear Controller Derivation

A continuous single-axis system may be described by,

$$\begin{aligned}\dot{\theta} &= \omega \\ \dot{\omega} &= \frac{F_{jet} \cdot r}{I_{yy}} = \alpha\end{aligned}\quad (1)$$

where  $r$  is the jet control moment arm,  $I_{yy}$  is the inertia term, and  $F_{jet}$  is the fixed-amplitude jet force. A discrete representation is given by,

$$\begin{aligned}\theta_T &= \theta_0 + \omega_0 \cdot T_{ACS} - \frac{1}{2} \alpha \cdot p^2 + \alpha \cdot p \cdot (T_{ACS} - t_1) \\ \omega_T &= \omega_0 + \alpha \cdot p,\end{aligned}\quad (2)$$

where  $T_{ACS}$  is the control cycle period,  $t_1$  marks the beginning of the jet pulse, and  $p$  is the pulse width,  $p = t_2 - t_1$ , where,  $t_2$  marks the end of the pulse width.

Figure-3 shows the states,  $\theta$  and  $\omega$  as a function of time for a jet pulse occurring over a particular time period,  $t_1$  to  $t_2$ . Note that the beginning of the pulse is not shown to coincide with the beginning of the control cycle, and it is this feature of both pulse width and pulse time that the nonlinear controller relies on to satisfy the two error criteria of  $\Delta\theta \rightarrow 0, \Delta\omega \rightarrow 0$ , where  $\Delta\theta = \theta_d - \theta_T$ ,  $\Delta\omega = \omega_d - \omega_T$  and,  $\theta_d$  and  $\omega_d$  are the desired end states.

The cost function is defined as,

$$J(t_1, p) = (\theta_d - \theta_T)^2 + k \cdot (\omega_d - \omega_T)^2 \quad (3)$$

which, as seen from Eq.-2, allows for two degrees of freedom,  $t_1$  and  $p$ , in order to satisfy the two criteria of  $\Delta\theta \rightarrow 0, \Delta\omega \rightarrow 0$ . The parameter  $k$  defines the relative weighting between the position and rate error. Note that because of the PWPT solution must lie within the next ACS control interval, a constrained optimization of Eq. (3) is needed.

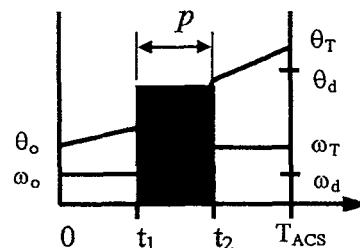


Figure-3: Discrete state transition over one ACS control cycle as a function of jet pulse width and jet pulse temporal position within the control cycle.

Referring to Figure-3, there are a total of five cases to be evaluated:

Case-1:  $t_1 \geq 0$  and  $t_2 \leq T_{ACS}$ , where,  $p = t_2 - t_1$

Case-2:  $t_1 < 0$  and  $t_2 \leq T_{ACS} \Rightarrow p = t_2$

Case-3:  $t_1 \geq 0$  and  $t_2 > T_{ACS} \Rightarrow p = T_{ACS} - t_1$

Case-4:  $\frac{\Delta\omega}{\alpha} \leq p_{\text{minimum pulse}} \Rightarrow \Delta\omega \equiv 0, t_1 = 0$

Case-5:  $t_1 < 0$  and  $t_2 > T_{ACS}$ , large angle saturation.

The cost function of Case-1 (depicted in Figure-3) is an unconstrained case and can be evaluated against  $t_1$  and  $p$  in order to derive expressions for their optimal values as follows,

$$\frac{\partial J}{\partial t_1} = 2 \cdot (\theta_d - \theta_T) \cdot \alpha \cdot p \equiv 0 \Rightarrow \theta_d = \theta_T, \text{ for } p \neq 0,$$

so that,

$$\begin{aligned}\Delta\theta &= -\frac{1}{2} \alpha \cdot p^2 + \alpha \cdot p \cdot (T_{ACS} - t_1) \\ t_1^* &= T_{ACS} - \left( \frac{p}{2} + \frac{\Delta\theta}{\alpha \cdot p} \right)\end{aligned}\quad (4)$$

We next seek the optimal value of  $p$  by taking the derivative of  $J$  w.r.t.  $p$ .



$$\begin{aligned}
\frac{\partial J}{\partial p} &= -2 \cdot (\theta_d - \theta_T) \cdot (T_{ACS} - t_1 - p) \\
&\quad + k \cdot (\omega_d - \omega_T) \equiv 0 \\
\Rightarrow \omega_d &= \omega_T, \text{ since } \theta_d = \theta_T \\
\Delta\omega &= \omega_d - \omega_0 = \alpha \cdot p \\
p^* &= \frac{\Delta\omega}{\alpha}
\end{aligned} \tag{5}$$

Case-2 implies noncausality, as it would be necessary to begin the jet pulse prior to the beginning of the current control cycle. Thus, for Case-2, the cost function is evaluated using a causal constraint by setting  $\dot{t}_1 = 0$ ,

$$\begin{aligned}
\frac{\partial J}{\partial p} &\equiv 0 \Rightarrow \\
(\theta_d - \theta_T) \cdot (T_{ACS} - p) - k \cdot (\omega_d - \omega_T) &\equiv 0,
\end{aligned}$$

which leads to,

$$\begin{aligned}
p^3 - 3 \cdot T_{ACS} \cdot p^2 \\
+ 2 \cdot \left( T_{ACS}^2 + k - \frac{\Delta\theta}{\alpha} \right) \cdot p \\
- \frac{2}{\alpha} \cdot (\Delta\theta \cdot T_{ACS} + k \cdot \Delta\omega) &= 0
\end{aligned} \tag{6}$$

Eq.-6 shows that it is necessary to solve a cubic polynomial; this will also be true of Cases-3 and -4.

Case-3 corresponds to a jet pulse that is requested to turn on during the current ACS period, yet extend beyond the period. A constraint is imposed by truncating the desired pulse so that it does not extend into the next control cycle (i.e.,  $t_2 = T_{ACS} \Rightarrow p = T_{ACS} - t_1$ ). Therefore in Case-3,  $t_1$  and  $p$  are linearly dependent on one another, allowing one degree of freedom in which to evaluate the cost function,

$$\begin{aligned}
\frac{\partial J}{\partial p} &\equiv 0 \Rightarrow p^3 + 2 \cdot \left( \frac{k \cdot \alpha - \Delta\theta}{\alpha} \right) \cdot p \\
- 2 \cdot \frac{k \cdot \Delta\omega}{\alpha} &= 0
\end{aligned} \tag{7}$$

Case-4 pertains to the condition of entering a control cycle with an angular rate error that would produce a pulse width request that is less than the minimum impulse bit of the jet. For practical purposes this implies that  $p = 0$ , and by extension,  $\Delta\omega \equiv 0$ . For Case-4 a choice is made to begin the valve command at the beginning of the control cycle,  $\dot{t}_1 = 0$ , which leads to the following equation for determining  $p^*$ ,

$$\begin{aligned}
\frac{\partial J}{\partial p} &\equiv 0 \Rightarrow p^3 - 3 \cdot T_{ACS} \cdot p^2 + 2 \cdot \left( T_{ACS}^2 + k - \frac{\Delta\theta}{\alpha} \right) \cdot p \\
- \frac{2}{\alpha} \cdot \Delta\theta \cdot T_{ACS} &= 0
\end{aligned} \tag{8}$$

Case-5 is the "large-angle" or saturated case in which a solution to the previous four "small-angle" solutions does not exist. Case-5 therefore merely serves to bring the system into any of Case-1 through Case-4, the cases of our interest. Thereafter, an undisturbed system will remain captured in Cases-1, -2, -3, or -4.

Solutions to the cubic equations are computed and those that are positive and real are used in the evaluation of the cost function. The root yielding the minimum-valued cost function is denoted as  $p^*$ .

It should finally be noted that a default "Case-0" exists wherein  $p^* < p_{\text{minimum pulse}}$  (i.e., the system state has reached equilibrium).

Figure-4 below illustrates the controller Cases, with the single-pulse switching line indicated by arrows (the axes have been normalized such that  $\theta_{\text{Max}} = \omega_{\text{Max}}$ , over  $\Delta T_{ACS}$ ). Case-1 includes the area in phase-space that can be reached during the current control cycle. Case-2 corresponds to the need to have pulsed prior to the current control cycle and then extend into the current. Case-3 corresponds to the need to first coast, then use a single pulse that extends beyond the current control cycle into the next. Case-4 represents  $\Delta\omega \equiv 0$ , and Case-5 represents the large-angle region.

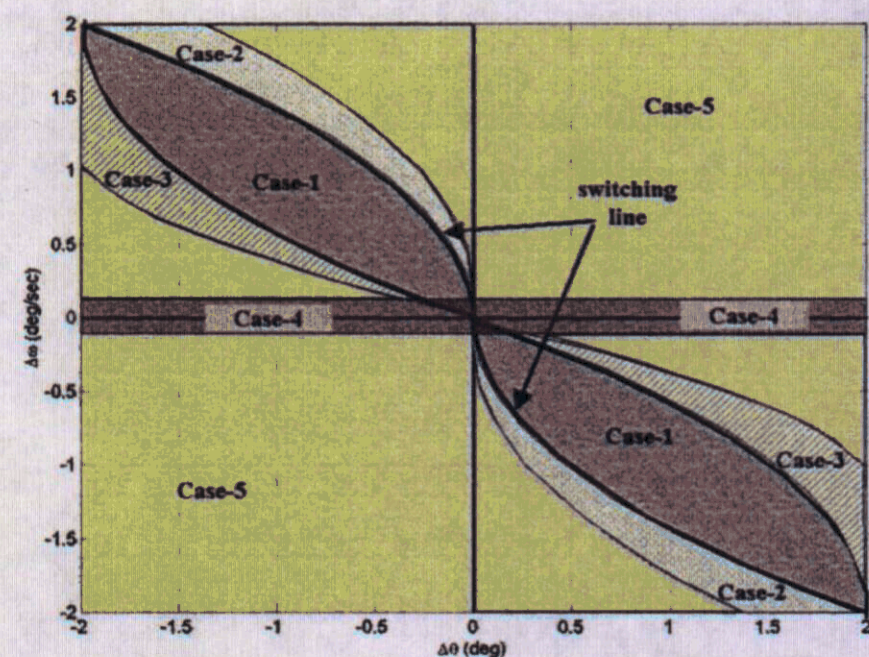


Figure-4: Controller cases shown in phase-space

## Single-Axis Simulation Results

The single-axis controller is applied in a deterministic setting (noiseless sensors, ideal actuator), and then the controller is applied in a stochastic setting in order to evaluate the controller's sensitivities to gyro and camera noise.



### Deterministic Scenario

A 1-DOF Simulink model was developed to evaluate the performance of the single-axis controller. An integration routine with fixed step size of  $\Delta T = \frac{1}{2} \text{ ms}$  was used and the simulations were run for 10 seconds. Other parameters included a minimum valve command of  $\frac{1}{2} \text{ ms}$ , pitch angular acceleration  $\alpha = 0.18 \text{ rad/s}^2$ , control cycle  $T_{ACS} = 25 \text{ ms}$ , and cost function weighting factor  $k = 2.5 \text{ s}^2$ . A nonzero initial angle and angle rate was applied to the system which was required to drive the final states  $[\theta_f \ \omega_f]^T$  to  $[0 \ 0]^T$ . Figure-5 shows the jet valve commands and Figure-6 shows the corresponding closeup of the valve commands for the system once it has been captured in the small-angle mode. Figure-7 illustrates the case transitions for the system in the small angle mode, Figure-8 shows the angle and angular rate errors, and the phase plane response is shown in Figure-9.

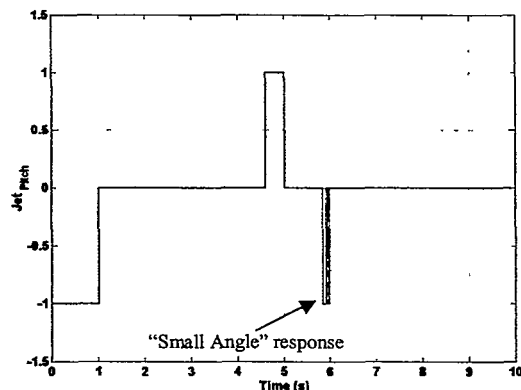


Figure-5: Jet valve commands for single-axis model

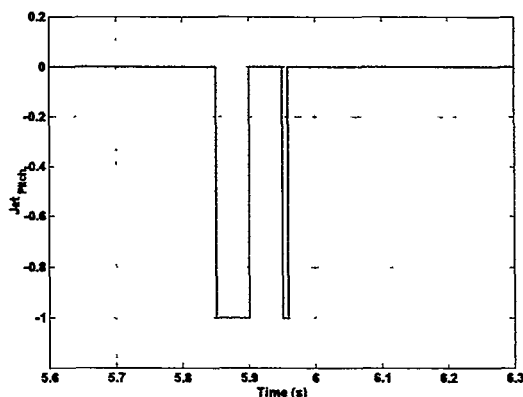


Figure-6: Closeup of Figure-5 ("Small Angle") on/off commands corresponding to Cases-1,2,3,4

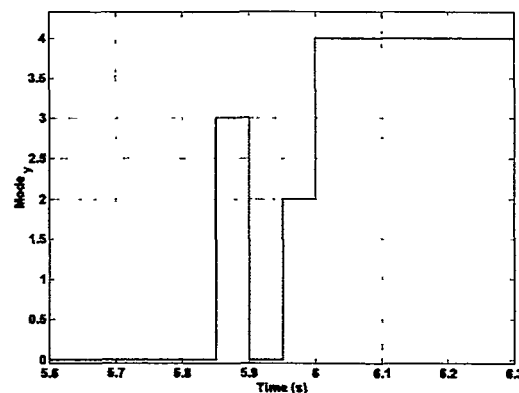


Figure-7: Closeup of case transitions for the "Small Angle" modes of Figure-5

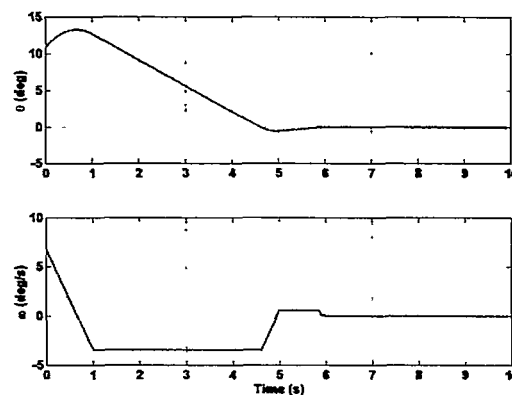


Figure-8: System state errors

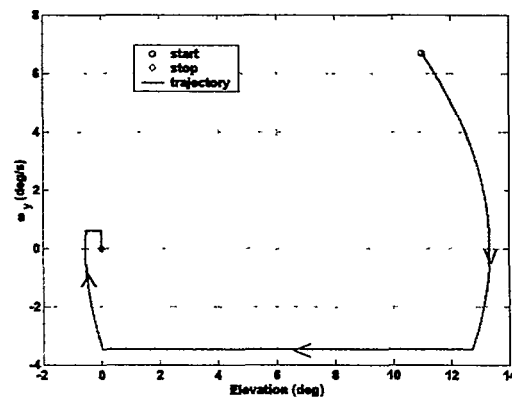


Figure-9: Phase-plane response of system

### Stochastic Scenario

In order to evaluate the controller in less than ideal conditions (e.g., model sensor noises) Monte Carlo runs ( $N=100$ ) were made with respect to gyro (rate) noise and camera (angle) noise, and the results

are compared against the baseline scenario. The gyro noise was chosen as a zero-mean normal distribution, with standard deviations of either  $50\mu\text{rad/s}$  (baseline),  $100\mu\text{rad/s}$ , or  $200\mu\text{rad/s}$ , and the camera noise was chosen as a zero-mean normal distribution with standard deviations of either  $100\mu\text{rad}$  (baseline),  $200\mu\text{rad}$ , or  $300\mu\text{rad}$ . Random draws were made on the initial states  $[\theta_0 \ \omega_0]^T$  of each run such that  $\theta_0 = N(0\text{rad}, (100\text{mrad})^2)$  and  $\omega_0 = N(0\text{rad/s}, (50\text{mrad/s})^2)$ , where  $N(\mu, \sigma^2)$  represents a normal distribution with mean  $\mu$  and standard deviation  $\sigma$ . The controller variables-of-interest are the final state errors  $([\Delta\theta_f \ \Delta\omega_f]^T)$  and the total number of pulse commands ( $p_{\text{MEDIAN}}$ ,  $p_{\text{MAXIMUM}}$ ). Figure-9 shows the gyro sensitivity curves and Figure-10 shows the camera sensitivity curves (all results are summarized in Table-1: Summary of Monte Carlo sensitivity studies).

From Figure-10 it is seen that the variations of the final angle error, due to gyro noise, decreased from a baseline value of  $0.07\text{mrad}$  to  $0.04\text{mrad}$  at maximum gyro noise due to the increased number of pulses (dithering due to increased noise). The angle rate error increased from a baseline value of  $0.04\text{ mrad/s}$  to  $0.32\text{ mrad/sec}$  at maximum gyro noise. The median and maximum number of pulses increased linearly where  $p_{\text{MEDIAN}}$  varied from a baseline value of  $8\text{pulses}$  at  $50\mu\text{rad/s}$  to  $98\text{pulses}$  at  $200\mu\text{rad/s}$ , and  $p_{\text{MAXIMUM}}$  varied from  $29\text{pulses}$  to  $152\text{pulses}$ .

Looking at the variations due to camera noise, Figure-11 shows that the angle error is roughly flat with respect to camera noise, whereas the rate error significantly increases with camera noise. Both  $p_{\text{MEDIAN}}$  and  $p_{\text{MAXIMUM}}$  fit well to a quadratic curve where  $p_{\text{MEDIAN}}$  increased from a baseline value of  $8\text{pulses}$  at  $100\mu\text{rad}$  to  $68\text{pulses}$  at  $300\mu\text{rad}$ , and  $p_{\text{MAXIMUM}}$  increased from  $29\text{pulses}$  to  $88\text{pulses}$ .

Conclusions for the 1-DOF simulation can be summarized as follows: Assuming a noise-free environment and assuming that the controller is operating in the small-angle region (Cases-1 through -4) it has been shown that the controller can take the system from an initial state to a final state in two or three pulses, depending on the initial point in angle-angle rate phase space. In an environment with noise present, the controller is least sensitive with respect to the final angle error, and exhibits considerable sensitivities with respect to the final angle rate error and the number of pulses required.

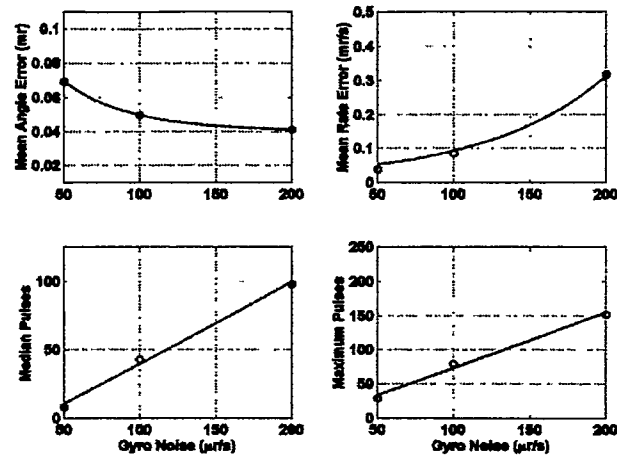


Figure-10: Monte Carlo 1-DOF sensitivities to gyro noise

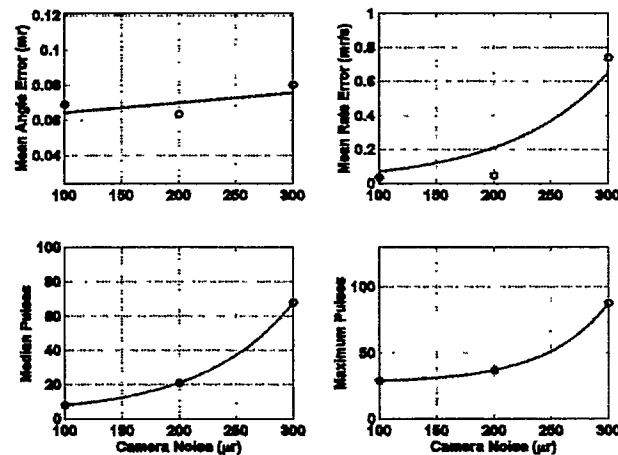


Figure-11: Monte Carlo 1-DOF sensitivities to camera noise

### Three-Axis Monte Carlo Simulation Results

A fully-coupled 3-DOF Simulink model was developed in order to evaluate the performance of the single-axis controller when applied to each of the three axes independently. Eq.-9 represents the dynamics and illustrates the cross-coupling effects that occur among axes. The inertia matrix was chosen to contain only moment-of-inertia terms, and six actuator jets were modeled as opposing pairs per axis. Other parameters remained the same as in the 1-DOF simulation (e.g., minimum valve commands of  $1.0\text{ ms}$ ). The pitch angular acceleration,  $\alpha_{\text{pitch}} = 0.18\text{ rad/s}^2$ , remained the same, and roll and yaw

angular accelerations were arbitrarily chosen as,  $\alpha_{roll} = 0.22 \text{ rad/s}^2$  and  $\alpha_{yaw} = 0.20 \text{ rad/s}^2$ .

$$\dot{\bar{Q}} = -\frac{1}{2}[\Omega] \cdot \bar{Q} \quad (9)$$

$$\dot{\bar{\omega}} = I^{-1} \cdot (\bar{\tau} - \bar{\omega} \times (I \cdot \bar{\omega}))$$

where,  $\bar{Q}$  is the attitude quaternion defined as

$\bar{Q} = [q_{scalar} \quad \bar{q}_{vector}]$ ,  $[\Omega]$  is a skew symmetric matrix comprised of elements of the angular rate vector,  $\bar{\omega}$ ,  $I$  is the inertia matrix and,  $\bar{\tau}$  is the applied jet torques.

As in the 1-DOF scenario, Monte Carlo runs ( $N=100$ ) were made with respect to gyro noise and camera noise, and the results are compared against the baseline scenario. The gyro noise was chosen as zero-mean Gaussian, with standard deviations of either  $50 \mu\text{rad/s}$  (baseline),  $100 \mu\text{rad/s}$ , or  $200 \mu\text{rad/s}$  and the camera noise was also chosen as zero-mean Gaussian with standard deviations of either  $100 \mu\text{rad}$  (baseline),  $200 \mu\text{rad}$ , or  $300 \mu\text{rad}$ . Random draws were made on the initial states  $[\theta_0 \quad \omega_0]^T$  of each run such that  $\theta_0 = N(0\text{rad}, (100\text{mrad})^2)$  and  $\omega_0 = N(0\text{rad/s}, (50\text{mrad/s})^2)$ , and the controller was required to drive the angles and angular rates to zero.

Figures-12 and -13 show the cumulative sensitivity results for gyro noise and camera noise, respectively. In this context, "cumulative", with respect to the angle errors and angular rate errors, is defined by the norm of the three-axis errors, whereas cumulative with respect to the number of pulses is defined by the sum of the three-axis pulses. The cumulative 3-DOF  $p_{\text{MEDIAN}}$  for the baseline scenario is 19 pulses which, when compared to the 1-DOF system's baseline scenario of 8 pulses, is slightly less than an ideal increase of a factor-of-three. Thus, it could be speculated that the coupling among axes is assumed to be responsible for the unexpected five fewer pulses in the 3-DOF system. Similarly, the cumulative 3-DOF  $p_{\text{MAXIMUM}}$  for the baseline scenario is 63 pulses which, when compared to the 1-DOF system's baseline scenario of 29 pulses, is again below the ideal increase of a factor-of-three. A similarly consistent pattern holds for the 3-DOF angle and angular rate errors as in the 1-DOF system.

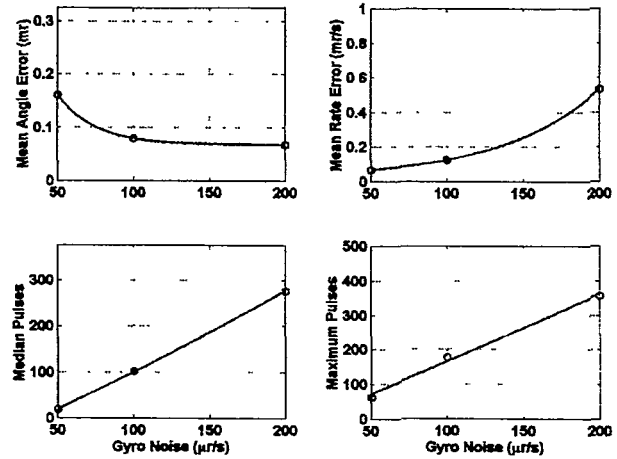


Figure-12: Monte Carlo 3-DOF cumulative sensitivities to gyro noise

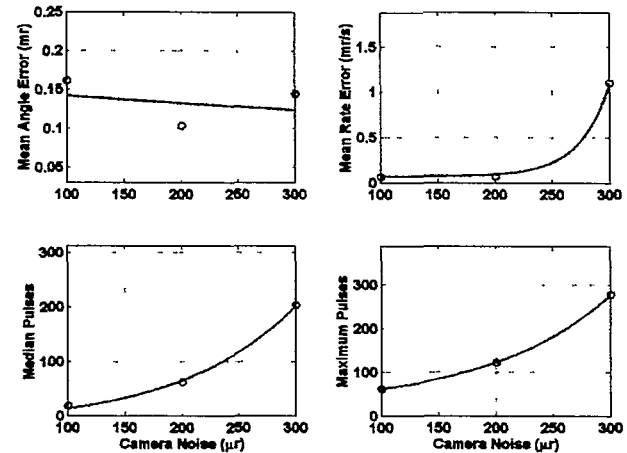


Figure-13: Monte Carlo 3-DOF cumulative sensitivities to camera noise

In the 3-DOF system the angle errors and angular rate errors are roughly larger by a factor of  $\sqrt{3}$  as compared to those in the 1-DOF system. Table-1 summarizes the results of the 1- and 3-DOF sensitivity studies for the nonlinear controller as well as results for a conventional PID controller (PID results are shown in parentheses). Similar higher-



Parameter	Variable	1-DOF Baseline $\sigma_{\text{Gyro}}=50\mu\text{rad/sec}$ $\sigma_{\text{Camera}}=100\mu\text{rad}$	1-DOF Max $\sigma_{\text{Gyro}}$ $\sigma_{\text{Gyro}}=200\mu\text{rad/sec}$ $\sigma_{\text{Camera}}=100\mu\text{rad}$	3-DOF Baseline $\sigma_{\text{Gyro}}=50\mu\text{rad/sec}$ $\sigma_{\text{Camera}}=100\mu\text{rad}$	3-DOF Max $\sigma_{\text{Gyro}}$ $\sigma_{\text{Gyro}}=200\mu\text{rad/sec}$ $\sigma_{\text{Camera}}=100\mu\text{rad}$
Gyro	$\Delta\theta_{\text{Final}}$ ( <i>mrad</i> )	0.07 (0.29*)	0.04 (0.18)	0.16 (1.42)	0.07 (1.05)
Gyro	$\Delta\omega_{\text{Final}}$ ( <i>mrad/s</i> )	0.04 (0.09)	0.32 (0.06)	0.07 (0.14)	0.54 (0.15)
Gyro	P <sub>MEDIAN</sub> ( <i>pulses</i> )	8 (32)	98 (28)	19 (100)	277 (108)
Gyro	P <sub>MAXIMUM</sub> ( <i>pulses</i> )	29 (51)	152 (40)	63 (126)	358 (136)

Parameter	Variable	1-DOF Baseline $\sigma_{\text{Gyro}}=50\mu\text{rad/sec}$ $\sigma_{\text{Camera}}=100\mu\text{rad}$	1-DOF Max $\sigma_{\text{Camera}}$ $\sigma_{\text{Gyro}}=50\mu\text{rad/sec}$ $\sigma_{\text{Camera}}=300\mu\text{rad}$	3-DOF Baseline $\sigma_{\text{Gyro}}=50\mu\text{rad/sec}$ $\sigma_{\text{Camera}}=100\mu\text{rad}$	3-DOF Max $\sigma_{\text{Camera}}$ $\sigma_{\text{Gyro}}=50\mu\text{rad/sec}$ $\sigma_{\text{Camera}}=300\mu\text{rad}$
Camera	$\Delta\theta_{\text{Final}}$ ( <i>mrad</i> )	0.07 (0.29)	0.08 (0.14)	0.16 (1.42)	0.14 (1.20)
Camera	$\Delta\omega_{\text{Final}}$ ( <i>mrad/s</i> )	0.04 (0.09)	0.74 (0.05)	0.07 (0.14)	1.10 (0.14)
Camera	P <sub>MEDIAN</sub> ( <i>pulses</i> )	8 (32)	68 (25)	19 (100)	204 (113)
Camera	P <sub>MAXIMUM</sub> ( <i>pulses</i> )	29 (44)	88 (57)	63 (126)	278 (146)

Table-1-a,b: Summary of nonlinear controller Monte Carlo sensitivity studies ( \* PID controller results shown in parentheses)

level patterns are seen in the 3-DOF and 1-DOF simulations, e.g., final angle errors decrease with increasing gyro noise due to dithering, angular rate errors increase quadratically for gyro and camera noise, median and maximum pulses are linear with respect to gyro noise and quadratic with respect to camera noise.

### Comparison of the Nonlinear and PID Controllers

Several observations can be extracted from Table-1 regarding the relative performance of the nonlinear controller with a conventional PID controller (used in this paper as a benchmark for comparison purposes). The first observation is that the nonlinear controller performs better, with low noise sensors (baseline scenario), than the PID controller for all four metrics considered ( $\Delta\theta_{\text{Final}}$ ,  $\Delta\omega_{\text{Final}}$ , P<sub>MEDIAN</sub>, P<sub>MAXIMUM</sub>). Therefore, for low noise applications (e.g., gyro noise =  $50\mu\text{rad/s}$ , and camera noise =  $100\mu\text{rad}$ ), the nonlinear controller provides better state control with fewer pulses. This is expected since the nonlinear controller solves for precise solutions required to remove state errors. However, at higher sensor noise levels (e.g., gyro

noise =  $200\mu\text{rad/s}$ , and camera noise =  $300\mu\text{rad}$ ), only the final angular state error,  $\Delta\theta_{\text{Final}}$ , is smaller for the nonlinear controller. Conversely, when regarding the final angular rate error,  $\Delta\omega_{\text{Final}}$ , and number of median and maximum actuator pulse commands, the PID controller shows superior performance in the high noise scenario (shown in *italics* in Tables 1-a,b).

### Summary, Conclusions, and Recommendations

It has been demonstrated that the optimal single-axis nonlinear controller, when applied to a noise-free 1-DOF model, eliminates the initial state errors to within the tolerance of the minimum pulse width of the jets and the nonideality introduced by two constraints in the controller's derivation. The first constraint occurred in Case-2, where the cost function was evaluated by setting  $\dot{x}_1=0$  (which led to Eq.-6). The second constraint occurred in Case-4, where the minimum impulse of the jet forced us to solve the cubic equation by using  $\Delta\omega \equiv 0$  (which led to Eq.-8). Irrespective of errors introduced by these variations on the ideal solution, the controller was

shown to require either two or three pulses to correct for initial errors, depending upon the initial phase-plane state.

Monte Carlo runs were made using a 1-DOF model in order to observe the sensitivity of the controller to gyro and camera noise, from which it was concluded that final angle errors decrease with increasing gyro noise due to dithering and angular rate errors increase quadratically for gyro and camera noise. The median and maximum pulses increase linearly with respect to gyro noise, and increase quadratically with respect to camera noise.

Monte Carlo runs were also made using a 3-DOF model in order to observe the sensitivity of the controller to gyro and camera noise, and to additionally observe how well the controller performed in the presence of a fully-coupled three-axis model. It was observed that the coupling among axes did not significantly change the characteristics of the controller's response, i.e., it behaved similarly as in the 1-DOF model with a linear scaling effect created by the fact that there were now three axes.

A comparison between the nonlinear controller and a conventional PID controller was made. It was concluded that the nonlinear controller performed better in a lower noise environment with respect to all four performance metrics, and also in the higher noise environment with respect to the final angular error. However, the PID controller performed better in a higher noise environment on three of the four metrics (final angular rate error, and median and maximum number of pulses). The Monte Carlo simulations have demonstrated the usefulness of the nonlinear controller in an application where the sensor (and by extension the actuator) noise characteristics are below some overall system-level-dependent threshold. An example of this kind of application is the requirement for precision pointing and station-keeping of an exo-atmospheric satellite.

One explanation for the seemingly poor performance of the PWPT controller, at very high angle and angular rate sensor noise, can be attributed to the fact that the cost function weighting factor,  $k$ , did not capture the full relationship between the sensor noise characteristics (i.e., sensor covariance matrix,  $R$ ). The cost function could have been reformulated as,

$$J = [(\theta_d - \theta_T) \quad (\omega_d - \omega_T)] \cdot R^{-1} \cdot [(\theta_d - \theta_T) \quad (\omega_d - \omega_T)]^T$$

$$\text{where, } R = \begin{bmatrix} \sigma^2_{\theta\theta} & \sigma^2_{\theta\omega} \\ \sigma^2_{\theta\omega} & \sigma^2_{\omega\omega} \end{bmatrix}.$$

We believe with the above new formulation of the cost function, the performance of the PWPT controller will be improved significantly.

It should be noted that this controller was derived to satisfy the requirement of taking the system from an initial to a final point on the phase plane. Thus, the controller does not explicitly address the issue of tracking a moving target. However, a predictive filter could be implemented that supplies the controller with nonstationary endpoints at each control cycle, thereby allowing the controller to track a moving target. Further studies should be conducted to quantify, for this more general scenario (i.e., a moving target), the performance of the single-axis controller relative to a conventional controller (e.g., PID, sliding mode).

Controller stability as a function of the weighting parameter,  $k$  (as shown in Eq.-3) should also be studied. We have not derived a Lyapunov-type stability proof. However, by inspection of Eq.-3 it is observed that the controller's damping is directly related to the weighting parameter,  $k$ , and that as  $k \rightarrow 0$  the system becomes marginally stable.

## References

- [1] Wertz, James R., Spacecraft Attitude Determination and Control, Kluwer Academic Publishers, 1978.
- [2] Wie, Bong, Space Vehicle Dynamics and Control, AIAA Education Series, 1998.
- [3] Garcia, Fermin, "A Non-linear Fuel Optimal Control Law for Spacecraft Attitude Control with Reaction Jets," MIT M.S. Thesis, 1998.